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#### Power Generation and Flame Visualization of Micro Gas Turbine Firing Ammonia or Ammonia-Methane Mixture

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- 1. Introduction
- 2. Experimental Apparatus
- 3. Results
- 4. Future task
- 5. Summary





# Image of NH3 Production and Use

- Japan starts the project of R&D focusing on energy carriers in 2013.
- Ammonia has become one of the attractive energy carriers.
- •This project includes ammonia production and ammonia utilization.
- AIST has been responsible for the demonstration of gas turbine firing ammonia to show the potential of ammonia as a fuel.
- As the first step of the project, AIST tried to demonstrate a small gas turbine firing ammonia gas with kerosene to obtain the knowhow concerning to ammonia gas handling and combustion of ammonia.







## Characteristics of NH3

- •Boiling point: -33.4°C
- Nonignitable fuel Vapor pressure: 1.177MPa (30°C)
- Vapor density: 0.76 g/l (gas, 0°C,0.1013MPa)
- Specific gravity: 0.59 (Air =1;25°C)
- Flammability: inflammable in air
- Auto-ignition temperature: 651°C
- Decomposition temperature: 840°C-930°C
- Explosion mixture limits : 15 28%
- Laminar Burning Velocity : 7cm/s at max.(NH3-Air Mixture) 1/5 of that of CH4-Air Mixture
- High NOx Emission
- Lower radiation than hydrocarbon flame

Source: Takachiho Chemical Industrial Co. Ltd, MSDS Sheet Ube Industries Ltd., MSDS sheet Hayasaka, et.al, 12<sup>th</sup> NH3 Fuel Conference, 2015





## Gas turbine firing ammonia

Project	year	
T-350 gas turbine (USA)	1960's	R&D of vapor combustor R&D of catalytic combustor Engine test
Space Propulsion Group Inc. (Sunnyvale, CA)	2010's	Jet engine test
Cardiff University	2010's	R&D of ammonia-methane combustor
IHI (SIP energy carrier)	2013-	R&D of methane-ammonia combustor

source:http://nh3fuelassociation.org/introduction/、Brian Evans, 10<sup>th</sup> NH3 Fuel Conference, Program of 12<sup>th</sup> NH3 Fuel Conference, Agustin Valera-Medinaa, et. al., Applied Energy, 2016.02.073





#### **R&D** of Gas Turbine Firing Ammonia in AIST

Phase I : NH3-Keosene Combustion FY 2013-2014



21kW power generation was achieved with about 30% decrease of kerosene by supplying ammonia gas.

Ammonia gas supply to the NOx removal equipment can decrease NOx emission very well.



Phase III : Combustor Test Rig CFD FY 2016 -

Start of combustion test by Combustor Test Rig



Council for Science, Technology and Innovation (CSTI), Crossministerial Strategic Innovation Promotion Program (SIP), "energy carrier" (Funding agency: JST).

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## Base Micro Gas Turbine

- A 50kW class micro gas turbine was selected as the base engine of ammonia fueled gas turbine.
- This gas turbine was made by TOYOTA TURBINE AND SYSTEMS INC (TTS).
  778.8



#### Specification of base micro gas turbine

Manufacturer	Toyota Turbine and System Inc. (TTS)
Cycle	Regenerative cycle
Shaft	Single shaft
Compressor	Centrifugal one-stage
Turbine	Radial one-stage
Rotating Speed	80,000rpm
Electric Power Output	50kW
Fuel	Kerosene
Combustor	Single can, Diffusion combustion





## Prototype combustor

- •The standard combustor is replaced by a prototype combustor.
- •Diffusion combustion is employed in the prototype combustor due to its flame stability.
- •Prototype combustor is designed for bifuel operation of kerosene and gas fuel.
- •A conventional swirl injector for kerosene is set in the center of the combustor inlet.
- •Gas fuel is supplied from 12 holes outside of the kerosene injector.
- •Recuperator can supply hot combustion air.
- 6 holes for bypass air are added on the combustor liner to reduce air velocity of around fuel injector and to increase the temperature of combustion gas.



Ammonia combustor

(prototype bi-fuel combustor)

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#### A test facilities for micro gas turbine power generation



- 50kW class gas turbine needs about 200kW of fuel input (17kg//h) in the case that 25% of thermal efficiency.
- Latent heat of vaporization is about 6% of higher heating value. Over 12kW of heat is necessary for evaporation of ammonia supply for 50kW class gas turbine.
- Hence planed ammonia supply facility includes 1 ton gas cylinder and vaporizer as shown in figure.
- Liquid ammonia is supplied from 1 ton gas cylinder and evaporates by vaporizer using hot water.
- Fuel gas compressor compresses ammonia gas to over 0.5MPa and compressed ammonia gas supplied to a gas turbine.
- Exhaust gas of a gas turbine passes through NOx removal equipment and is emitted.
- A methane gas supply facility is also prepared for basic research of natural gas-ammonia co-firing. As the first step, methane gas is mixed to pressurized ammonia gas near the gas fuel inlet the prototype combustor.

Test facilities for micro gas turbine power generation

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scale \_\_\_\_ 1000





## Starting-up procedure



- Because the ignition of pure NH<sub>3</sub> was difficult (ignition energy is high), ignition by kerosene firing was adopted during the start-up procedure.
- After starting up, gas-fuel (NH<sub>3</sub>, CH<sub>4</sub>) was supplied gradually replacing kerosene, then the ratio of gas-fuel to kerosene was increased until the kerosene supply was stopped.
- Finally, gas-fuel-air combustion was established, and the test conditions of the NH<sub>3</sub>-air and NH<sub>3</sub>-CH<sub>4</sub>-air combustion gas turbine were set.



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### Operation range for NH<sub>3</sub>-air combustion



- The operation was successfully performed over the electric power range of 18.4 kW to 44.4 kW, and for rotating speeds ranging from 70000 rpm to 80000 rpm.
- There are two limits. The limit is not due to instability of  $NH_3$ -air combustion. The higher power limit is due to high temperature of material. The lower power limit is due to unburnt  $NH_3 > NOx$ .
- The SCR NOx reduction apparatus enables an increased concentration of unburnt NH<sub>3</sub> until it exceeds the concentration of NOx.

![](_page_12_Picture_6.jpeg)

![](_page_13_Picture_0.jpeg)

![](_page_13_Figure_1.jpeg)

- NO emission increases dramatically with an increase of NH<sub>3</sub> ratio.
- NO emission have a peak concentration.
- However, from a certain point, unburnt NH<sub>3</sub> increases and NO decreases as NH<sub>3</sub> ratio increases further.
- This is due to NO reduction before SCR. It is expected that Selective Non-Catalytic Reduction (SNCR) occurred.

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![](_page_14_Picture_0.jpeg)

## NOx Emission and NOx Removal

![](_page_14_Figure_2.jpeg)

![](_page_14_Figure_3.jpeg)

![](_page_14_Figure_4.jpeg)

#### **Outline of NOx removal process**

- NOx removal equipment employs SCR (Selective Catalytic Reduction) method.
- Ammonia gas was added to the exhaust gas for NOx removal.
- NOx, here summation of NO and NO<sub>2</sub>, is about 700ppm and O<sub>2</sub> was 18.3%.
- Ammonia gas added to the exhaust gas decrease NOx emission.
- When added ammonia gas to the exhaust gas was 700ppm, its amount was only about 2% of total ammonia gas.

![](_page_14_Picture_11.jpeg)

![](_page_15_Picture_0.jpeg)

### Flame observation

![](_page_15_Picture_2.jpeg)

- The adaptor for the flame observation from the combustor exit was prepared.
- The adaptor is the bent coaxial tubes with the quartz window.

Combustor

![](_page_15_Picture_5.jpeg)

Quartz

Window

Turbine &

Turbine

Inlet

Flame observation

Compressor

![](_page_15_Figure_6.jpeg)

![](_page_15_Picture_8.jpeg)

![](_page_16_Picture_0.jpeg)

# Flame observation

Ignition and Kerosene Flame at Starting-up of Gas Turbine

![](_page_16_Picture_3.jpeg)

240fps image ,1/8 slow

Ammonia Flame

(75,000rpm,39.1kW output, Monochrome)

#### **240fps image ,1/8 slow** Methane-Ammonia Flame (LHV Ratio 1:1)

Methane-Ammonia Flame (75,000rpm,39.1kW output, Monochrome)

#### 240fps image ,1/8 slow

![](_page_16_Picture_10.jpeg)

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![](_page_17_Picture_0.jpeg)

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![](_page_17_Picture_7.jpeg)

![](_page_18_Picture_0.jpeg)

## Future task

Near future

• Flame observation under the operation

Future task and under planning

- Flame observation and measurement with a combustor test rig
- Modification of combustor (fuel injector, combustor linear, etc.)
- Development of low NOx combustor
- Starting up of a gas turbine firing methane or natural gas

![](_page_18_Picture_9.jpeg)

![](_page_19_Picture_0.jpeg)

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![](_page_19_Picture_7.jpeg)

![](_page_20_Picture_0.jpeg)

# Summary

50kW class micro gas turbine firing kerosene was remodeled for power generation firing ammonia.

A standard combustor is replaced with a prototype combustor which enables a bi-fuel supply of kerosene and gas fuel. Adapter for flame observation was prepared.

[Prototype combustor]

- The operation firing ammonia was successfully performed over the electric power range of 18.4 kW to 44.4 kW, and for rotating speeds ranging from 70000 rpm to 80000 rpm.
- There are two limits. The limit is not due to instability of  $NH_3$ -air combustion. The higher power limit is due to high temperature of material. The lower power limit is due to unburnt  $NH_3 > NOx$ .

[With adapter flame observation]

Flame observation was succeeded while the prototype gas turbine generated electric power.

![](_page_20_Picture_9.jpeg)

![](_page_21_Picture_0.jpeg)

# Thank you for your attention !!

#### Acknowledgement

- This work was supported by Council for Science, Technology and Innovation (CSTI), Cross-ministerial Strategic Innovation Promotion Program (SIP), "energy carrier" (Funding agency: JST).
- The authors also thank to Mr. Imura, Mr. Okada, Ms. Namatame, and "Toyota Turbine and Systems Inc." for the advice on the combustion technology and the operation of micro gas turbine.

![](_page_21_Picture_5.jpeg)